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Prior Foreign Application(s)

外国での先行出願

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2001-26586	Japan
(Number)	(Country)
(番号)	(国名)
2000-345459	Japan
(Number)	(Country)
(番号)	(国名)

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優先権主張なし 2/February/2001 (Day/Month/Year Filed) (出願年月日) 8/November/2000 (Day/Month/Year Filed)

(出願年月日)

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(現況:特許許可済、係属中、放棄済)

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委任状: 私は下記の発明者として、本出願に関する一切の 手続きを米特許商標局に対して遂行する弁理士または代理人 として、下記の者を指名いたします。(弁護士、または代理 人の氏名及び登録番号を明記のこと) POWER OF ATTORNEY: As a named inventor, I hereby appoint the following attorney(s) and/or agent(s) to prosecute this application and transact all business in the Patent and Trademark Office connected therewith (list name and registration number)

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subsequent joint inventors.)

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ratio can be obtained by engaging assist mechanism 140 when running by using the 3rd gear. Therefore, the decrease in the number of the 3rd drive gears becomes possible in this case, and the number of the parts can be decreased. Further, lightening can be obtained as a result, the inertia of the body of revolution can be reduced, and the load to the clutch can be decreased. [0106]

Next, the transmission according to the eighth embodiment of the present invention will be explained with reference to Fig.21. Fig.21 is a view showing the whole configuration of the automatic transmission according to an eighth embodiment of the present invention. Fig.21 is another embodiment in which the transmission is used for the front engine rear drive.

[0107]

Fig.21 shows the example in which the transmission for

the front engine front drive of Fig. 19 is applied for the front engine rear drive. In this example, assist output gear 131 and assist input gear 130 arranged in assist axis 104 is arranged to engage with a gear provided on 1st axis (input shaft) 102. Also in this case, the torque from 1st axis 102 (input shaft) to the 2nd axis 103(output shaft) while shifting is transferred through a gear (2nd drive gear 112 in Fig.20) engaged with 1st axis (input shaft) 102, a gear (2nd driven gear 122 in Fig.20) which runs idle freely with respect to 2nd axis (counter axix) 103 engaged said gear, assist input gear 130 provided on assist axis 104, assist output gear 131, assist mechanism 140, and 3rd driven gear 123 engaged with the 2nd axis (counter axis) 103. In this case, even when there is no room of arrangement in the lower part of the transmission, it is possible to apply as well as the case in the FF vehicle of Fig.19. In addition, the miniaturization of transmission can be achieved by sharing reverse idol axis and assist axis 104.

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[0108]

The transmission according to the ninth embodiment of the present invention will be explained with reference to Fig.22. Fig.22 is another embodiment in which motor generator 200 is installed.

[0109]

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In Fig.22, motor generator 200 is provided in the transmission according to the embodiment of Fig.18 and 20 previously shown. In this transmission, the gear with the same number of teeth and diameter as 3rd drive gear is applied to assist output gear 131 arranged on assist axis 104. 3rd drive gear 113 arranged on 1st axis (input shaft) 102 is removed. The torque transfer while shifting is as has already described. Further, the running of 3rd gear becomes the same state as the running state of the 3rd speed gear by engaging assist mechanism 140. Therefore, the 3rd drive gear can be removed by adopting the configuration shown in Fig.22, and the lightening of transmission can be obtained as a result, the inertia of the body of revolution can be reduced, and the cost can be reduced.

[0110]

There is an effect that when the torque transfer from 1st axis 102(input shaft) to the 2nd axis 103(output shaft)while shifting is done by the 3rd gear gear ratio, the torque transfer route while shifting and the torque transfer route of 3rd gear under running can be shared, and the number of gears can be suppressed to the minimum. Oppositely, there is an effect that by setting separately the torque transfer route while shifting and the torque transfer route while normally running, the free setting of the amount of the torque transfer while shifting becomes possible.

[0111]